# CFD MODEL TO ANALYZE SO<sub>2</sub> ABSORPTION IN A SEAWATER DROPLET

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### SUMMARY

In the recent years, seawater scrubbers have become an interesting option to reduce  $SO_2$  emissions in marine engines. In this regard, this paper proposes a numerical model to analyze  $SO_2$  absorption in seawater. A single seawater droplet was analyzed, and the developed model was used to predict the influence of several parameters on the desulphurization efficiency, such as the droplet diameter,  $SO_2$  concentration, alkalinity and temperature. It was found that a droplet of 200  $\mu$ m initial diameter can absorb up to  $1.77 \cdot 10^{-14}$  mol of S for the parameters analyzed, and this reduction improves when the alkalinity and  $SO_2$  concentration are increased and diameter, seawater temperature and gas temperature are reduced. Differences up to of 21.5%, 19.8%, 2.2% and 16.3% in the S reduction were obtaining varying the  $SO_2$  initial concentration, initial liquid temperature and initial gas temperature respectively.

#### NOMENCLATURE

A	Area (m <sup>2</sup> )
$\Delta G^0$	Standard Gibbs free energy change (J/mol)
D	Diameter (m or µm)
З	Small quantity (m)
$\phi$	Level set (m)
h	Heat transfer coefficient (W m <sup>-2</sup> K)
k	Henry's constant (mol kg <sup>-1</sup> atm <sup>-1</sup> )
Pr	Prandtl number (-)
R	Universal gas constant (J mol <sup>-1</sup> K <sup>-1</sup> )
Re	Reynolds number (-)
ρ	Density (kg m <sup>-3</sup> )
σ	Surface tension (N m <sup>-1</sup> )
ξ	Artificial time (s)
Т	Temperature (K)
t	Time (s)
и	Velocity (m s <sup>-1</sup> )

v Kinematic viscosity (N s m<sup>-2</sup>)

#### 1. INTRODUCTION

Although most combustion systems have achieved a significant reduction of their emissions during the last years, it is necessary to further reduce them due to the increasingly restrictive legislation. Particularly, in the marine field most medium and large marine engines operate on heavy fuel oil, which is a cheap combustible but contains an important quantity of pollutant substances, especially sulphur (S) (Lamas, et al, 2013). The oxidation of sulphur in the fuel forms sulphur oxides (SO<sub>x</sub>) in the exhaust gas, especially SO<sub>2</sub> (Lamas, et al, 2015; Lamas & Rodriguez, 2017). SO<sub>x</sub> are the mayor source of acid rain (Lamas & Rodriguez, 2013), which causes breathing difficulties, respiratory illnesses, acidification of lakes, agricultural crops, corrosion of materials, etc. In order to prevent these consequences, the International Maritime Organization (IMO) established a set of regulations to reduce emissions. Particularly, Annex VI of MARPOL 73/78 regulates the SO<sub>x</sub> emissions by setting a maximum limiting value on the fuel sulphur content. This limit has been reduced from 4.5% to 3.5% from 1 January 2012 and then from 3.5 to 0.5% w/w from 1 January 2020. Besides, there are sulphur emission control areas (SECAs) with more stringent requirements. The sulphur limits and implementation dates required by IMO are illustrated in Figure. 1. Instead of using low sulphur content fuels, other allowed solution is to employ exhaust gas cleaning/after-treatment systems. These have become a promising alternative due to the high price of low sulphur fuels such as Marine Diesel Oil and Marine Gas Oil (Seddiek & Elgohari, 2014).



Figure 1. Sulphur limits and implementation dates.

The most common method for flue gas desulphurization is scrubbing through an alkali absorbent like urea (Barbooti, et al, 2011), NaOH (Hikita, et al, 1977), limestone slurry (Lancia, et al, 1997; Puskar, et al, 2013), NaCl (Jeong & Kim, 1997), water (Bokotko, et al, 2005) and seawater, which is a promising solution in marine applications. The main advantages of seawater are the availability and the fact that the acidified effluent can be discharged directly into the sea after a simple neutralization process (Pyszko, et al, 2015; Ammar & Seddiek, 2017). There are some reports about seawater scrubbers available in the literature. For instance, Zhang et al (1988), Oikawa *et al.* (2003) and Williams (1999) analyzed seawater wet scrubbers used in power plants. Sun *et al.* (2008) showed that the mass transfer coefficient in seawater is about twice as great as that of the NaOH solution with pH 8.35. Darake *et al.* (2014) carried out an experimental study and mathematical modeling about SO<sub>2</sub> removal by seawater in a packedbed tower. Caiazzo *et al.* (2013) analyzed a seawater spray scrubber operating under marine diesel exhaust conditions and compared seawater to distilled water. Sukheon and Nishida (2003) analyzed the effect of seawater under SO<sub>x</sub>, particulate matter, CO<sub>2</sub> and NO<sub>x</sub>.

The complexity of the SO<sub>2</sub> absorption process makes it difficult to analyze through theoretical or experimental analyses. In the recent years, the development of computer technology has become numerical procedures an important tool to predict the droplet absorption process. Most of the numerical work available in the literature refers to simulations of the whole geometry of scrubbers. Instead the whole scrubber, the analysis of a single droplet provides information about diffusion, convection inside the droplet, or evaporation. There are several papers that analyze single liquid droplets in the literature, most of them focused on SO<sub>2</sub> scavenging by rainwater in the atmosphere. For instance, Babolal et al. (1981) studied atmospheric SO<sub>2</sub> scavenged by water drops by means of a simple numerical method. Huckaby and Ray (1989) analyzed numerically the absorption of SO<sub>2</sub> by a stationary water droplet under condensation or evaporation and, more recently, Chen et al. (Chen, 2001 & 2006, Chen, et al, 2011 and Chen, et al, 2012) developed a CFD model to analyze SO<sub>2</sub> reduction by water droplets in the atmosphere analyzing both liquid droplets and gas medium.

The purpose of this paper is to develop a numerical model to analyze  $SO_2$  absorption by seawater and continue the work developed elsewhere (Lamas et al, 2016). To this end, a moving water droplet immersed in a gas medium was analyzed. Both liquid and gas phases were included in the model. The proposed CFD model takes in consideration the fluid motion inside and outside the droplet, heat transfer, chemical reactions and mass diffusion.

#### 2. NUMERICAL IMPLEMENTATION

The model proposed in the present paper analyzes the amount of  $SO_2$  absorbed by a single droplet which falls counter flow with respect to an exhaust gas stream with  $SO_2$ . The computational domain is indicated in Figure. 2. As can be seen, the domain size is  $15 \times 6$  droplet radii from the droplet center since it was verified that these dimensions eliminate any potential effect of the outer boundaries on the flow close to the droplet. Several tests were performed in order to set the more appropriate dimension of the computational domain. The problem was simulated as axisymmetric.



Figure 2. Computational domain.

When viewed in a fixed coordinate system, the droplet and gas move relatively with each other. Modeling this movement requires a large domain and thus high computational time. An alternative approach is to work using a coordinate system which moves with the droplet. In this case, the droplet is fixed in space and the relative droplet-gas velocity is imposed as inlet and outlet boundary conditions, *i.e.*, the gas flow enters and leaves the computational domain at the relative velocity. This velocity is variable due to the effects of buoyancy and drag. Buoyancy tends to accelerate the droplet, while drag tends to decelerate it.

### 2.1 CHEMICAL MODEL

The principle of operation of a seawater scrubber is based on a spray of droplets which fall counter flow to an exhaust gas containing SO<sub>2</sub>. When the gas is in touch with a droplet, the concentration at the interface follows the Henry's law. A concentration gradient is induced in the interface and SO<sub>2</sub> is transported into the droplet by mass diffusion according to the Fick's law. Chemical dissociation reactions also take place inside the droplet.

While several alkaline species are presented in seawater ( $\text{HCO}_3^-$ ,  $\text{CO}_3^{2-}$ ,  $\text{B}(\text{OH})_4^-$ ,  $\text{OH}^-$ ,  $\text{HPO}_4^{2-}$ , etc), the main contribution to alkalinity is by far the bicarbonate ion  $\text{HCO}_3^-$ , and typical concentrations are 2,400µmol/kg of H<sub>2</sub>O (Sverdrup, et al, 1942; Dickson & Goyet, 1994 & Andreason & Mayer, 2007).

Regarding chemical reactions, the model proposed by Andreasen and Mayer (2007) was adopted. This assumes the following chemical reactions:

$SO_2(g) \leftrightarrow SO_2(aq)$	(1)
50 2(g) ( ) 50 2 (uq)	(-)

- $SO_2(aq) + 2H_2O(l) \leftrightarrow HSO_3^-(aq) + H_3O^+(aq)$  (2)
- $HSO_{3}^{-}(aq) + H_{2}O(l) \leftrightarrow SO_{3}^{2-}(aq) + H_{3}O^{+}(aq)$ (3)
- $HCO_{3}^{-}(aq) + H_{3}O^{+}(aq) \leftrightarrow CO_{2}(aq) + 2H_{2}O(l)$ (4)
- $\operatorname{CO}_2(\operatorname{aq}) \leftrightarrow \operatorname{CO}_2(\operatorname{g})$  (5)

Reaction (1) represents the dissolution of gaseous SO<sub>2</sub> in water at the liquid-gas interphase, and it is governed by Henry's law:

$$[SO_2(aq)] = p_{SO_2}k_H \tag{6}$$

where  $p_{SO2}$  is the partial pressure of SO<sub>2</sub>, [SO<sub>2</sub>(aq)] the concentration of SO<sub>2</sub> in the solution and  $k_H$  the Henry's constant, expressed as (Sander, 2005):

$$k_{H} = k_{H}^{0} e^{\frac{-\Delta H_{\text{soln}}}{R} \left(\frac{1}{T} - \frac{1}{T^{0}}\right)}$$
(7)

where  $k_{H}^{0}$  is the Henry's constant at the reference state,  $\Delta H_{soln}$  the enthalpy of solution, *T* the temperature in Kelvin and  $T^{0}$  the reference state temperature (298.15 K). A value of  $k_{H}^{0} = 1.2 \text{ mol}/(\text{kg atm})$  and the slope -  $\Delta H_{soln}/R$ = 2850 K were employed, Sander (2005).

Reaction (5) represents the dissolution of  $CO_2$  at the liquidgas interphase. This is also governed by Henry's law.

Reactions (2-4) occur inside the droplet and their kinetics was assumed to be in equilibrium (Chen, et al, 2011 & Andreason, 2007). The equilibrium condition was imposed by minimizing the Gibbs free energy according to the mass action law. Reactions (2-4) can be written in general form as follows:

$$aA + bB \leftrightarrow cD + dD$$
 (8)

where A and B are reactants, C and D products, and a, b, c, and d the stoichiometric coefficients. According to the mass action law, the Gibbs free energy is minimum when:

$$-\frac{\Delta G_0}{RT} = ln \left( \frac{[\mathbf{C}]^c[\mathbf{D}]^d}{[\mathbf{A}]^a[\mathbf{B}]^b} \right)$$
(9)

where  $\Delta G^0$  is the standard Gibbs free energy change, *R* the universal gas constant, [A] the molar concentration of species A at equilibrium, and so forth. The term  $[C]^d[D]^d/([A]^a[b]^b)$  is the so-called equilibrium constant for the reaction.

#### 2.2 GOVERNING EQUATIONS

The governing equations are the transport equations of conservation of mass, momentum and energy, Eqs. (10-12) respectively. A laminar simulation was carried out due to the low droplet diameters and velocities employed. Inside the droplet, the energy equation was solved until the evaporation temperature was reached. From this temperature, it was imposed as constant and a diameter reduction due to evaporation was implemented in the code.

$$\nabla \cdot \boldsymbol{u} = \boldsymbol{S}_{\boldsymbol{w}} \tag{10}$$

$$\rho \frac{\partial \vec{u}}{\partial t} + \rho \nabla \cdot (\vec{u}\vec{u}) = -\nabla p + \rho \vec{g} - \nabla \cdot \tau + S_{mo}$$
(11)

$$\rho c \frac{\partial T}{\partial t} + \rho c \nabla \cdot (\vec{u}T) = k \nabla^2 T + S_e$$
(12)

In the equations above, u is the velocity,  $\rho$  the density, t the time, p the pressure,  $\tau$  the stress tensor, c the specific heat and k the thermal conductivity. The terms  $S_m$ ,  $S_{mo}$  and  $S_e$  are sources in the equations of mass, momentum and energy respectively.

The source term  $S_m$  of Eq. (10) accounts for the phase change, and is given by (Sim, et al, 2015):

$$S_m = \left(\frac{1}{\rho_v} - \frac{1}{\rho_l}\right)m\tag{13}$$

where  $m = \nabla Hh(T_e - T_g) / h_h$  and the term  $h(T_e - T_g)$ represents the heat transferred between the droplet and gas. Therefore,  $h(T_e - T_g)/h_h$  is the mass transfer per unit area along the interface.  $T_e$  is the evaporation temperature,  $T_g$  the gas temperature,  $h_h$  the latent heat of evaporation, and h the heat transfer coefficient, given by the Ranz-Marshall equation (Ranz & Marshall, 1952):

$$\frac{hD}{k} = 2 + 0.6 \,\mathrm{Re}^{1/2} \,\mathrm{Pr}^{1/3} \tag{14}$$

where k is the thermal conductivity, Re the Reynolds number, Pr the Prandtl number and D the droplet diameter. In the present work the shape of the droplet remained practically spherical due to the importance of the surface tension force in comparison with the inertial force.

H is the so-called Heaviside function. In the present paper, this takes a 0 value for the liquid, 1 for the vapor and a value between 0 and 1 along the interface, Eq. (15).

$$H = \begin{cases} 0 & \text{if } \phi < \varepsilon \\ 1 & \text{if } \phi > \varepsilon \\ (\phi + \varepsilon)/(2\varepsilon) + [\sin(\pi \phi / \varepsilon)]/(2\pi) & \text{if } |\phi| \le \varepsilon \end{cases}$$
(15)

where  $\varepsilon$  is a small quantity of the order of the mesh size.

The source term  $S_e$  indicated in Eq. (12) also accounts for the phase change, and is given by:

$$S_e = h_{lv} m \tag{16}$$

The source term  $S_{mo}$  of Eq. (11) accounts for the effect of surface tension, given by (Brackbill, et al, 1992):

$$S_{mo} = \sigma \kappa \nabla H \tag{17}$$

where  $\kappa$  is the curvature of the interface.

The interface was treated using the level set method. This procedure consists on assigning a scalar magnitude, the level set  $\phi$ , to the distance from the interface. The interface corresponds to a zero value of the level set. In the present work, a negative sign of  $\phi$  was adopted for the vapor and positive for the liquid. The propagation of the interface is provided by the following equation:

$$\frac{\partial \phi}{\partial T} + \vec{u}_{\text{int}} \cdot \nabla \phi = 0 \tag{18}$$

Where  $\vec{u}_{int}$  is the velocity of the interface, which can be obtained by a mass and heat transfer balance at the interface, given by:

$$\vec{m} = \frac{h(T_e - T_g)}{h_{tv}} = \rho(\vec{u}_{int} - \vec{u})$$
(19)

In order to preserve the property of being distance from the interface, the level set function must be reinitialized at each time step through the following equation:

$$\frac{\partial \phi}{\partial \xi} = sign(\phi_0)(1 - |\nabla \phi|) \tag{20}$$

where  $\xi$  is an artificial time introduced because Eq. (20) is a temporal equation which must be solved at each time step, and  $\phi_0$  the level set field at the start of each iteration in this artificial time, *i.e.*:

$$\phi(x,0) = \phi_0(x) \tag{21}$$



Figure 3. Bubble radius applied to a spherical water bubble growing in a 5°C superheated liquid medium.

The accuracy of the evaporation model was validated with experimental results elsewhere (Lamas, 2012; Lamas, et al, 2012; and Lamas, et al, 2015). Several authors reported results about spherical bubble growing due to evaporation (Plesset & Zwick, 1954; Scriven, 1959; Mikic, et al, 1970; and Foster & Zuber, 1954) and developed correlations based on their experimental findings. As comparison between these correlations proposed in the literature and the results proposed by the present numerical model, Figure. 3 shows the evolution of the bubble radius against time for a spherical bubble of water immersed in a liquid medium 5°C superheated. As can be seen in this figure, the developed numerical model provides reasonable results.

#### 2.3 BOUNDARY CONDITIONS

As mentioned previously, a reference frame which moves with the droplet was employed. A free-stream velocity was imposed as inlet and outlet boundary conditions. The drag force decelerates the relative velocity between the droplet and gas, while buoyancy accelerates it. This was implemented into the numerical model by adjusting the free-stream velocity at each time step. The acceleration or deceleration is given by:

$$\frac{du_{\infty}(t)}{dt} = g - \frac{F_{drag}}{\rho_{l}\pi D^{3}/6}$$
(22)

where  $u_{\infty}$  is the free-stream velocity and  $F_{drag}$  the drag force, given by:

$$F_{drag} = \frac{C_D A_f \rho_g (u_{\infty} + u_g)^2}{2}$$
(23)

where  $u_g$  is the gas velocity,  $A_f$  the reference area (for a sphere  $\pi D^2/4$ ), and  $C_D$  the drag coefficient, obtained by the following expression (Seinfeld, 1986):

$$C_D = \frac{24}{\text{Re}} \left( 1 + 0.15 \,\text{Re}^{0.687} \right) \tag{24}$$

Finally, the variation of the free-stream velocity during a time step  $\Delta t$  is:

$$u_{\infty}(t + \Delta t) = u_{\infty}(t) + \Delta t \frac{du_{\infty}(t)}{dt}$$
(25)

#### 2.4 CFD MODEL

The computational mesh is indicated in Figure. 4. The problem was simulated using the open software OpenFOAM (Open Field Operation and Manipulation). This software was chosen because it allows a total manipulation of the code. A new OpenFOAM solver was programmed for the present study. C++ programming language was employed to write it.



Figure 4. Computational mesh.

As indicated previously, several tests were performed in order to determine the adequate extent of the numerical domain in such a way to eliminate any potential effect of the outer boundaries (theoretically at infinity) on the flow close to the droplet. In addition, the solution was checked for refinement sensibility on both the mesh size and time step. Four meshes and time steps were tested to verify that the results are independent of these parameters. The number of elements of these meshes was 1414 (Mesh 1), 1821 (Mesh 2), 2112 (Mesh 3), and 2412 (Mesh 4), while the time steps were  $2.5 \cdot 10^{-4}$  s (Time step 1),  $5 \cdot 10^{-5}$  s (Time step 2),  $10^{-5}$  s (Time step 3), and  $2 \cdot 10^{-6}$  s (Time step 4). The SO<sub>2</sub> absorbed by a droplet is indicated in Table 1. As can be seen, the results using Mesh 3 and Time step 3 are appropriate as refining mesh size or time step does not modify the error obtained. Therefore, Mesh 3 and Time step 3 were chosen for the present work.

Table 1. S absorbed by a droplet (mol/droplet) using several mesh sizes and time steps. 200  $\mu$ m initial droplet diameter, 600 ppm SO<sub>2</sub> concentration, 2,400 $\mu$ mol/kg alkaline concentration, 400°C initial gas temperature and 20°C initial droplet temperature.

	Time step 1	Time step 2	Time step 3	Time step 4
Mesh 1	1.62.10-14	$1.67 \cdot 10^{-14}$	1.69.10-14	1.73.10-14
Mesh 2	1.65.10-14	1.73.10-14	$1.77 \cdot 10^{-14}$	$1.77 \cdot 10^{-14}$
Mesh 3	1.73.10-14	$1.77 \cdot 10^{-14}$	$1.77 \cdot 10^{-14}$	$1.77 \cdot 10^{-14}$
Mesh 4	$1.71 \cdot 10^{-14}$	$1.77 \cdot 10^{-14}$	$1.77 \cdot 10^{-14}$	1.77.10-14

The pressure-velocity coupling was treated using the PISO (Pressure Implicit Splitting of Operators) procedure. The equations were discretized by the QUICK interpolation and the temporal treatment was solved by an implicit method.

#### 3. RESULTS

The numerical model developed in the present work was employed to study the influence of several parameters such as the droplet diameter, SO<sub>2</sub> concentration, alkalinity and temperature. The ranges analyzed correspond to typical concentrations obtained in diesel engine exhaust when burning heavy fuel oil (Kuiken, 2008; Woodyard, 2009; Lamas & Rodriguez, 2012; and Lamas & Rodriguez, 2018). Results are indicated below.

# 3.1 EFFECT OF THE INITIAL DROPLET DIAMETER

Figure. 5 shows the mass fraction of  $HCO_3^-$  overlaid with the velocity field for 0.5s using 600ppm SO<sub>2</sub> concentration (0.00133 kg of SO<sub>2</sub>/kg of H<sub>2</sub>O), 2,400µmol/kg alkaline concentration (0.000146 kg of alkaline/kg of H<sub>2</sub>O), free stream initial velocity 2m/s (1 m/s liquid droplet velocity and 1m/s gas flow velocity), 400°C initial gas temperature, 20°C initial liquid temperature and 1,000µm initial droplet diameter. This figure shows that an internal vortex is formed due to drag and  $HCO_3^-$  is consumed first at the interphase and then at the core of the droplet. SO<sub>2</sub> first accumulates at the droplet surface and then is transported from the interface to the core of the droplet by three mechanisms: radial diffusion, internal circulation and chemical reactions.



Figure 5. Mass fraction of  $HCO_3^-$  overlaid with velocity field.

Figure. 6 indicates the droplet diameter reduction for 1,000, 500 and  $200\mu m$  initial diameters. As expected, all droplets experiment a reduction of the diameter due to the evaporation effect implemented in the model. The time was represented between 0 and 3 seconds because

this is a reasonable value of residence time in scrubbers (Lamas, et al, 2016). The droplet velocity and position for these initial diameters are indicated in Figures. 7 and 8 respectively. Figures. 6-8 correspond to 600ppm SO<sub>2</sub> concentration, 2,400µmol/kg alkaline concentration, free stream initial velocity 2m/s, 400°C initial gas temperature and 20°C initial droplet temperature. As expected, lower diameters correspond to lower velocities. As indicated above, two effects are responsible for the droplet trajectory: buoyancy and drag. Larger droplets are associated to high gravity forces and thus to high velocities. Nevertheless, as the droplet diameter is reduced, buoyancy becomes less important and velocity tends to decay. As indicated below, 1 m/s was assumed for the droplet and 1 m/s for the gas. Figure. 7 indicates that all droplets depart from this initial velocity of 1 m/s and the largest droplet increments this velocity due to the importance of buoyancy, while the droplets of 500 and 200µm reduce this initial velocity due to the importance of drag.



Figure 6. Droplet diameter against time. 600 ppm SO<sub>2</sub> concentration, 2,400 µmol/kg alkaline concentration, 400 °C initial gas temperature and 20 °C initial droplet temperature.



Figure 7. Droplet velocity against time. 600 ppm SO<sub>2</sub> concentration, 2,400 µmol/kg alkaline concentration, 400°C initial gas temperature and 20°C initial droplet temperature.



Figure 8. Droplet position against time. 600 ppm SO<sub>2</sub> concentration, 2,400 µmol/kg alkaline concentration, 400°C initial gas temperature and 20°C initial droplet temperature.

The quantity of sulphur (S) absorbed by each droplet is indicated in Figure. 8. The absorbed S was computed as the sum of moles of  $SO_2$ ,  $HSO_3^-$  and  $SO_3^{2^-}$ . As can be seen, larger droplets are associated to higher S absorption rates. Nevertheless small droplets are associated to lower S absorption rates. The reason is simply that the volume is lower. The main advantage of small droplets is that these reach the saturation state in less time due to a more efficient contact with the liquid. In practical applications, this fact promotes small scrubbers.



Figure 9. S absorbed by a droplet for several initial diameters. 600ppm SO<sub>2</sub> concentration, 2,400µmol/kg alkaline concentration, 400°C initial gas temperature and 20°C initial droplet temperature.

# 3.2 EFFECT OF THE SO<sub>2</sub> INITIAL CONCENTRATION

Figure. 10 indicates the S absorbed by a droplet against time for several initial  $SO_2$  concentrations. This figure corresponds to 200 $\mu$ m initial droplet diameter, 2,400 $\mu$ mol/kg alkaline concentration, free stream initial

velocity 2m/s, 400°C initial gas temperature and 20°C initial droplet temperature. Three initial SO<sub>2</sub> concentrations were analyzed: 1,000, 750 and 600ppm. As can be seen, high SO<sub>2</sub> concentrations promote S absorption, and a 21.5% increment in the absorbed S was obtained from 600ppm to 1,000ppm.



Figure 10. S absorbed by a droplet for several SO<sub>2</sub> initial concentrations. 200µm initial droplet diameter, 2,400µmol/kg alkaline concentration, 400°C initial gas temperature and 20°C initial droplet temperature.

#### 3.3 EFFECT OF THE ALKALINITY

Figure. 11 indicates the S absorbed by a droplet against time for several alkaline concentrations. This figure corresponds to 200 $\mu$ m initial droplet diameter, 600ppm SO<sub>2</sub> concentration, free stream initial velocity 2m/s, 400°C initial gas temperature and 20°C initial droplet temperature. As can be seen, as alkalinity is increased SO<sub>2</sub> absorption is increased too, and a 19.8% increment in the absorbed S was obtained from 2,400 $\mu$ mol/kg to 5,000  $\mu$ mol/kg.



Figure 11. S absorbed by a droplet for several initial alkalinities. 200µm initial droplet diameter, 600ppm SO<sub>2</sub> concentration, 400°C initial gas temperature and 20°C initial droplet temperature.

#### 3.4 EFFECT OF THE TEMPERATURE

Figure. 12 indicates the S absorbed by a droplet against time for several initial droplet temperatures. This figure corresponds to 200 $\mu$ m initial droplet diameter, 2,400 $\mu$ mol/kg alkaline concentration, 600ppm SO<sub>2</sub> concentration, free stream initial velocity 2m/s and 400°C initial gas temperature. As can be seen, S absorption is improved when the droplet temperature is decreased, and a 2.2% variation in the absorbed S was obtained from 20°C to 60°C. The reason is that the concentration of SO<sub>2</sub> at the interphase is higher due to a higher value of the Henry's constant, indicated by Eq. (7).



Figure 12. S absorbed by a droplet for several initial droplet temperatures.  $200 \mu m$  initial droplet diameter, 2,400 \mu mol/kg alkaline concentration, 600 ppm SO<sub>2</sub> concentration and 400°C initial gas temperature.

Figure. 13 indicates the influence of the gas temperature. As can be seen, higher gas temperatures reduce S absorption, and a 16.3% variation in the absorbed S was obtained from 400°C to 500°C. The effect is more important than raising the droplet temperature. The reason is that higher gas temperatures induce higher interphase temperatures and thus lower  $SO_2$  concentrations at the interphase.



Figure 13. S absorbed by a droplet for several initial gas temperatures.  $200\mu m$  initial droplet diameter, 2,400 $\mu mol/kg$  alkaline concentration, 600ppm SO<sub>2</sub> concentration and 20°C initial droplet temperature.

## 4. CONCLUSIONS

This paper proposes a numerical model to analyze SO<sub>2</sub> absorption in seawater. SO2 scrubbers allow employing high-sulphur fuels with a low impact on sulphur emissions. In marine applications, seawater is a promising solution due to its availability. This paper proposes a numerical model to analyze SO<sub>2</sub> absorption in seawater. A single seawater droplet was analyzed, and the developed model was used to predict the influence of several parameters on the desulphurization efficiency, such as the droplet diameter, SO<sub>2</sub> concentration, alkalinity and temperature. Several tests were carried out using typical exhaust conditions of marine engines. It was found that a droplet of 200 µm initial diameter can absorb 1.77.10<sup>-14</sup> mol of S for the parameters analyzed, and this reduction improves when the alkalinity and SO<sub>2</sub> concentration are increased and diameter, seawater temperature and gas temperature are reduced. Differences up to of 21.5%, 19.8%, 2.2% and 16.3% in the S reduction were obtaining varying the SO<sub>2</sub> initial concentration, alkaline initial concentration, initial liquid temperature and initial gas temperature respectively.

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